

Reissner型中厚板弯曲理论 中的守恒积分

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摘 要

基于虚功原理导出受弯,中等厚度的,Reissner型板的,与路径无关的 J_R 积分。该积分可作为通常二维 J 积分的一般情形。在线弹性情形下,还得到了 J_R 与弯曲板应力强度因子 K 之间的简单关系式。当横向剪切刚度趋于无穷大,则 J_R 变为经典薄板的 J 积分。

一、引 言

自从Rice^[1]在平面问题中提出 J 积分以后,近年来已经有了广泛的应用,特别是在探讨弹塑性断裂理论方面起了较大的作用,在工程界板是常用的一种重要元件,随着断裂力学研究的不断深入,板的断裂分析也日益受到广泛的注意,鉴于 J 积分对平面问题富有成效,把类似的守恒积分推广到板的问题中来,也将是一项重要的工作,由于在板的断裂分析中,克希霍夫经典薄板理论有着重大缺陷,因此本文考虑Reissner^[2]型中厚板弯曲理论。本文从虚功原理出发给出了一个与路线无关的积分 J_R ,在线弹性的情况下,可以求出 J_R 与板的弯曲应力强度因子 K_B 的简单关系。在特殊情况下,此式可以退化为薄板经典理论中的 J 积分。本文给出的守恒积分在板的弹塑性理论中,可能会有重要的作用。在工程断裂的近似计算中,将会发挥较好的效果。

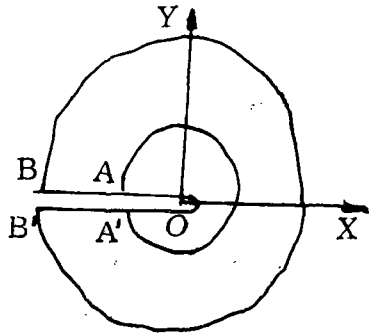
二、 J_R 的积分式和证明

在Reissner型中厚板的理论中,虚功原理^[3]为

$$\delta \int_{\Omega} \left[M_x \frac{\partial \phi_x}{\partial x} + M_{xy} \left(\frac{\partial \phi_x}{\partial y} + \frac{\partial \phi_y}{\partial x} \right) + M_y \frac{\partial \phi_y}{\partial y} \right] dx dy + \dots$$

$$\begin{aligned}
 & + \iint_{\Omega} \left[Q_x \left(\frac{\partial w}{\partial x} - \phi_x \right) + Q_y \left(\frac{\partial w}{\partial y} - \phi_y \right) \right] dx dy = \iint_{\Omega} (m_x \phi_x + m_y \phi_y + \\
 & + Pw) dx dy + \int_C (Q_n w - M_n \phi_n - M_{ns} \phi_s) dS \quad (1)
 \end{aligned}$$

其中内力 M_x 、 M_y 、 M_{xy} 、 Q_x 、 Q_y 应与外载荷保持平衡,即应满足平衡方程,而广义位移 ϕ_x 、 ϕ_y 、 w 只要连续、一次可导即可。广义位移和广义力之间可以没有任何联系, Ω 是板内的任一区域, C 为 Ω 的边界。



取笛卡尔坐标如图所示,原点在裂纹顶端, x 轴平行裂纹而指向物体内部,板在边界上受力而产生弯曲,即设

$$m_x = m_y = P = 0$$

在裂纹面上:

$$M_y = 0, M_{xy} = 0, Q_y = 0 \quad (2)$$

设问题的解是 ϕ_x 、 ϕ_y 、 w 、 M_x 、 M_y 、 M_{xy} 、 Q_x 、 Q_y 。

下列广义位移

$$\phi_x^K = \frac{\partial \phi_x}{\partial x}, \quad \phi_y^K = \frac{\partial \phi_y}{\partial y}, \quad w^K = \frac{\partial w}{\partial x} \quad (3)$$

是一种连续的位移,所以根据公式(1)有

$$\begin{aligned}
 & \iint_{\Omega} \left[-M_x \frac{\partial \phi_x^K}{\partial x} - M_{xy} \left(\frac{\partial \phi_x^K}{\partial y} + \frac{\partial \phi_y^K}{\partial x} \right) - M_y \frac{\partial \phi_y^K}{\partial y} + Q_x \left(\frac{\partial w^K}{\partial x} - \phi_x^K \right) + \right. \\
 & \left. + Q_y \left(\frac{\partial w^K}{\partial y} - \phi_y^K \right) \right] dx dy = \int_C (Q_n w^K - M_n \phi_n^K - M_{ns} \phi_s^K) dS \quad (4)
 \end{aligned}$$

这里 Ω 是板内的任一区域, C 为 Ω 的边界,现在取 C 为图中所示的 $A'B'BAA'$,因为 $A'B'$ 与 BA 为不受力的自由边,所以(4)式的右端化为

$$\begin{aligned}
 \text{右端} &= \int_{B'}^B (Q_n w^K - M_n \phi_n^K - M_{ns} \phi_s^K) dS + \int_{A'}^{A''} (Q_n w^K - M_n \phi_n^K - M_{ns} \phi_s^K) dS \\
 &= \int_{B'}^B (Q_n w^K - M_n \phi_n^K - M_{ns} \phi_s^K) dS - \int_{A'}^{A''} (Q_n w^K - M_n \phi_n^K - M_{ns} \phi_s^K) dS \quad (5)
 \end{aligned}$$

又根据(3)式,有

$$-M_x \frac{\partial \phi_x^K}{\partial x} - M_{xy} \left(\frac{\partial \phi_x^K}{\partial y} + \frac{\partial \phi_y^K}{\partial x} \right) - M_y \frac{\partial \phi_y^K}{\partial y} + Q_x \left(\frac{\partial w^K}{\partial x} - \phi_x^K \right) + Q_y \left(\frac{\partial w^K}{\partial y} - \phi_y^K \right)$$

$$\begin{aligned}
 &= \frac{\partial U'}{\partial K_x} \cdot \frac{\partial K_x}{\partial x} + \frac{\partial U'}{\partial K_{xy}} \cdot \frac{\partial K_{xy}}{\partial x} + \frac{\partial U'}{\partial K_y} \cdot \frac{\partial K_y}{\partial x} + \frac{\partial U''}{\partial \gamma_x} \cdot \frac{\partial \gamma_x}{\partial x} + \frac{\partial U''}{\partial \gamma_y} \cdot \frac{\partial \gamma_y}{\partial x} \\
 &= \frac{\partial}{\partial x} (U' + U'') = \frac{\partial U}{\partial x} \quad (6)
 \end{aligned}$$

式中 U 为应变能密度, U' 为弯曲应变能密度, U'' 为剪切应变能密度, 广义应变 K_x , K_y , K_{xy} , γ_x , γ_y 分别为

$$\begin{aligned}
 K_x &= -\frac{\partial \phi_x}{\partial x}, \quad K_y = -\frac{\partial \phi_y}{\partial y}, \quad K_{xy} = -\frac{1}{2} \left(\frac{\partial \phi_x}{\partial y} + \frac{\partial \phi_y}{\partial x} \right) \\
 \gamma_x &= \frac{\partial w}{\partial x} - \phi_x, \quad \gamma_y = \frac{\partial w}{\partial y} - \phi_y \quad (7)
 \end{aligned}$$

因此, (4)式的左端为

$$\text{左端} = \iint_C \frac{\partial U}{\partial x} dx dy = \int_C IU dS = \int_{B'} IU dS - \int_{A'} IU dS \quad (8)$$

从(6), (8)两式得到

$$\begin{aligned}
 &\int_{B'} IU dS - \int_{B'} (Q_n w^K - M_n \phi_n^K - M_{ns} \phi_s^K) dS \\
 &= \int_{A'} IU dS - \int_{A'} (Q_n w^K - M_n \phi_n^K - M_{ns} \phi_s^K) dS \quad (9)
 \end{aligned}$$

此式表示上列积分与路径无关, 把它记为 J_R , 则有

$$J_R = \int_C (IU - Q_n w^K + M_n \phi_n^K + M_{ns} \phi_s^K) dS$$

$$\text{即} \quad J_R = \int_C \left(IU - Q_n \frac{\partial w}{\partial x} + M_n \frac{\partial \phi_n}{\partial x} + M_{ns} \frac{\partial \phi_s}{\partial x} \right) dS \quad (10)$$

与路径无关, 只要 C 为由裂纹下面绕端点到正面一圈。

三、 J_R 与弯曲应力强度因子 K_B 关系

在Reissner型理论中, 裂纹顶端内力和位移的首项为

$$\begin{aligned}
 M_x &= \frac{h^2}{6} \left[\frac{K_{BI}}{\sqrt{2\pi\gamma}} \cos \frac{\theta}{2} \left(1 - \sin \frac{\theta}{2} \sin \frac{3\theta}{2} \right) - \frac{K_{BI}}{\sqrt{2\pi\gamma}} \sin \frac{\theta}{2} \left(2 + \cos \frac{\theta}{2} \cos \frac{3\theta}{2} \right) \right] \\
 M_y &= \frac{h^2}{6} \left[\frac{K_{BI}}{\sqrt{2\pi\gamma}} \cos \frac{\theta}{2} \left(1 + \sin \frac{\theta}{2} \sin \frac{3\theta}{2} \right) + \frac{K_{BI}}{\sqrt{2\pi\gamma}} \sin \frac{\theta}{2} \cos \frac{\theta}{2} \cos \frac{3\theta}{2} \right] \\
 M_{xy} &= \frac{h^2}{6} \left[\frac{K_{BI}}{\sqrt{2\pi\gamma}} \sin \frac{\theta}{2} \cos \frac{\theta}{2} \cos \frac{3\theta}{2} + \frac{K_{BI}}{\sqrt{2\pi\gamma}} \cos \frac{\theta}{2} \left(1 - \sin \frac{\theta}{2} \sin \frac{3\theta}{2} \right) \right]
 \end{aligned}$$

$$Q_x = -\frac{2h}{3} \cdot \frac{K_{BI}}{\sqrt{2\pi\gamma}} \sin \frac{\theta}{2} \quad (11)$$

$$Q_y = \frac{2h}{3} \cdot \frac{K_{BI}}{\sqrt{2\pi\gamma}} \cos \frac{\theta}{2}$$

$$\phi_x = -\frac{K_{BI}}{2Gh} \sqrt{\frac{\gamma}{2\pi}} \left[(2K-1) \cos \frac{\theta}{2} - \cos \frac{3\theta}{2} \right] - \frac{K_{BI}}{2Gh} \sqrt{\frac{\gamma}{2\pi}} \left[(2K+3) \sin \frac{\theta}{2} + \sin \frac{3\theta}{2} \right]$$

$$\phi_y = -\frac{K_{BI}}{2Gh} \sqrt{\frac{\gamma}{2\pi}} \left[(2K+1) \sin \frac{\theta}{2} - \sin \frac{3\theta}{2} \right] - \frac{K_{BI}}{2Gh} \sqrt{\frac{\gamma}{2\pi}} \left[-(2K-3) \cos \frac{\theta}{2} - \cos \frac{3\theta}{2} \right]$$

$$w = \frac{8K_{BI}}{5G} \sqrt{\frac{\gamma}{2\pi}} \sin \frac{\theta}{2}$$

其中 $K = \frac{3-\nu}{1-\nu}$

当(10)式中的C取得十分靠近裂纹顶端时,内力和位移便足够精确地用(11)式来表示。

将(11)式代入(10)式,算出积分,便可把 J_R 与 K_{BI} , K_{BI} , K_{BI} 联系起来,经一些运算后,可得

$$\int_C IU dS = \frac{h(1-\nu)}{12E} (K_{BI}^2 - K_{BI}^2)$$

$$\int_C \left(-Q_n \frac{\partial w}{\partial x} + M_n \frac{\partial \phi_n}{\partial x} + M_s \frac{\partial \phi_s}{\partial x} \right) dS = \frac{hK_{BI}^2}{12E} (3+\nu) + \frac{hK_{BI}^2}{12E} (5-\nu) + \frac{4h}{15G} K_{BI}^2$$

则

$$J_R = \frac{h}{3E} (K_{BI}^2 + K_{BI}^2) + \frac{4h}{15G} K_{BI}^2 \quad (12)$$

当考虑经典板的情况,即

$$\phi_n = \frac{\partial w}{\partial n}, \quad \phi_s = \frac{\partial w}{\partial s} \quad (13)$$

这时,

$$\begin{aligned}
 I &= \int_C \left[IU + \left(-Q_n \frac{\partial w}{\partial x} - M_n \frac{\partial \phi_n}{\partial x} + M_{ns} \frac{\partial \phi_s}{\partial x} \right) \right] dS \\
 &= \int_C \left[IU + \left(-Q_n \frac{\partial w}{\partial x} - M_n \frac{\partial^2 w}{\partial n \partial x} + M_{ns} \frac{\partial^2 w}{\partial S \partial x} \right) \right] dS \\
 &= \int_C \left[IU + \left(-Q_n \frac{\partial w}{\partial x} - M_n \frac{\partial^2 w}{\partial n \partial x} - \frac{\partial M_{ns}}{\partial S} \cdot \frac{\partial w}{\partial x} \right) \right] dS \\
 &= \int_C \left\{ IU + \left[- \left(Q_n + \frac{\partial M_{ns}}{\partial S} \right) \frac{\partial w}{\partial x} + M_n \frac{\partial^2 w}{\partial n \partial x} \right] \right\} dS \quad (14)
 \end{aligned}$$

(14)式为经典薄板理论的守恒积分形式。

参 考 文 献

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A Conservative Integral for Reissner Type Moderate Thick Plate in Bending

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ABSTRACT

Based on the virtual work principle, a path-independent integral J_R for the Reissner type moderate thick plate in bending is obtained. It can be regarded as a generalization of the conventional 2D J integral. In the linear elastic case a simple relationship between J_R and the bending stress intensity factor K is also obtained. When transverse shear stiffness tend to infinity, J_R is reduced to the J integral for the classical thin plate.